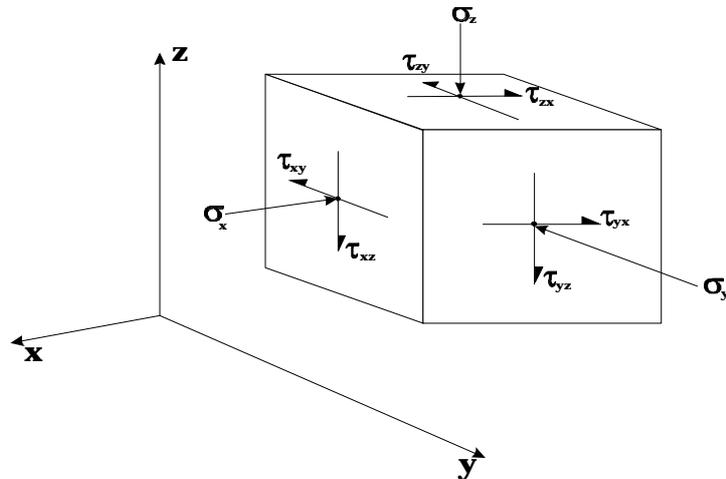




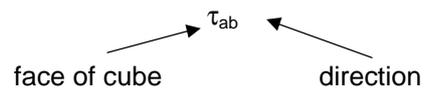
## **Continuum Mechanics**

# THEORY OF STRESSES

In a three dimensional loaded body, there are six independent components of stress at a point

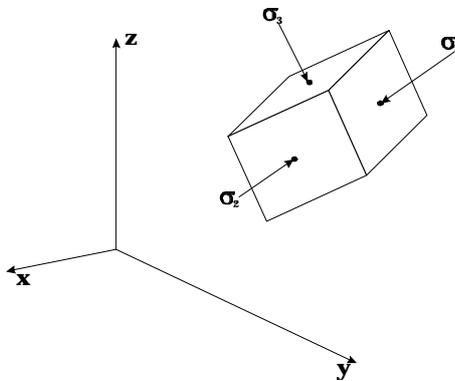


Shear stress notation:



By moment equilibrium:

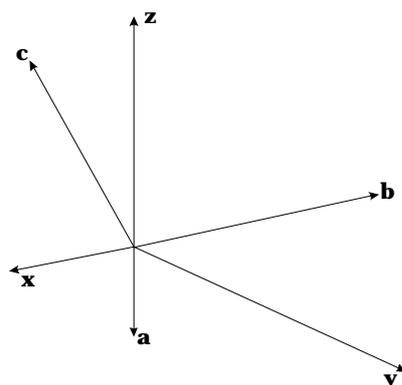
$$\tau_{xy} = \tau_{yx}, \quad \tau_{yz} = \tau_{zy}, \quad \tau_{xz} = \tau_{zx}$$



In any stressed material there are always three mutually perpendicular planes on which the shear stresses are zero.

These are called the principal planes.

The direct stresses acting on these planes are called principal stresses ( $\sigma_1, \sigma_2, \sigma_3$ ).



In a stressed body the components of stress are  $\sigma_x, \sigma_y, \sigma_z, \tau_{xy}, \tau_{yz}$  and  $\tau_{zx}$ .

We can also describe the stress state with respect to another set of axes (e.g. a, b, c).

Stress invariants are functions of the stress components which are independent of the axis system chosen. For example,

$$p = \frac{s_x + s_y + s_z}{3} = \frac{s_a + s_b + s_c}{3}$$

is a stress invariant.

Another stress invariant is q where:

$$q = \sqrt{\frac{(s_x - s_y)^2 + (s_y - s_z)^2 + (s_z - s_x)^2 + 6t_{xy}^2 + 6t_{yz}^2 + 6t_{zx}^2}{2}}$$

$$= \sqrt{\frac{(s_a - s_b)^2 + (s_b - s_c)^2 + (s_c - s_a)^2 + 6t_{ab}^2 + 6t_{bc}^2 + 6t_{ca}^2}{2}}$$

Note that these general definitions of p and q reduce to those given later for triaxial stress conditions (see lecture on Cam-clay).

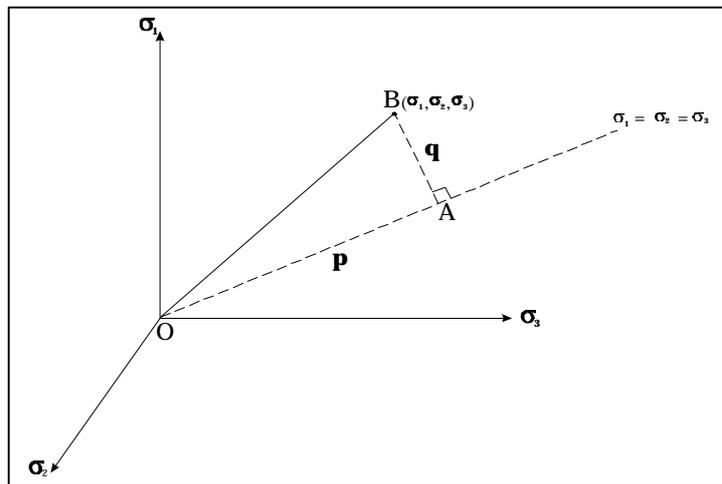
We will encounter these invariants more frequently expressed in terms of the principal stresses, i.e.

$$p = \frac{s_1 + s_2 + s_3}{3}$$

$$q = \sqrt{\frac{(s_1 - s_2)^2 + (s_2 - s_3)^2 + (s_3 - s_1)^2}{2}}$$

Note  $\sigma_1, \sigma_2, \sigma_3$  are also stress invariants.

There is a simple geometric interpretation of p and q in principal stress space.



The line  $\sigma_1 = \sigma_2 = \sigma_3$  is called the “space diagonal” or the “hydrostatic axis”

$$OA = \frac{s_1 + s_2 + s_3}{\sqrt{3}} = \sqrt{3} p$$

$$AB = \sqrt{\frac{2}{3}(s_1^2 + s_2^2 + s_3^2 - s_1 s_2 - s_2 s_3 - s_3 s_1)}$$

$$AB = \sqrt{\frac{2}{3}} q$$

# ELASTICITY

Generalised Hooke's Law:

$$e_x = \frac{s_x}{E} - \frac{ns_y}{E} - \frac{ns_z}{E}$$

$$e_y = -\frac{ns_x}{E} + \frac{s_y}{E} - \frac{ns_z}{E}$$

$$e_z = -\frac{ns_x}{E} - \frac{ns_y}{E} + \frac{s_z}{E}$$

$$g_{xy} = \frac{2(1+n)}{E} t_{xy}$$

$$g_{yz} = \frac{2(1+n)}{E} t_{yz}$$

$$g_{zx} = \frac{2(1+n)}{E} t_{zx}$$

# PLASTICITY

## INTRODUCTION

The theory of elasticity allows the calculation of stresses and strains in a loaded body when the body is linear and elastic.

The theory of plasticity allows the calculation of stresses and strains in a loaded body when plastic yielding takes place.

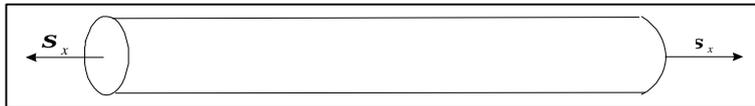
## History (Elasticity)

- |      |  |
|------|--|
| 1822 | Cauchy invents concepts of stress and strain   |
| 1885 | Boussinesq gives his solutions for elastic stress distributions (still used in geotechnical engineering today)<br><br><i>(For 100 years mathematicians slave away obtaining elasticity solutions...)</i> |
| 1956 | Finite element method invented (mathematicians are now redundant)  |

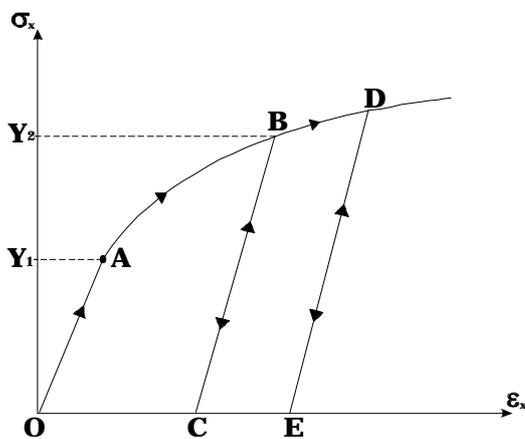
## History (Plasticity)

1773	Coulomb identifies two components in the strength of soil - cohesion and friction
1864 (Tresca)	Criteria are put forward for limits to the elastic behaviour of <u>metals</u>
1913 (Von Mises)	
1940's	3 types of statement are now seen to be necessary to completely describe plastic stress-strain relations:  a) yield criteria b) flow rule c) hardening law
1950	R. Hill's book "Mathematical Theory of Plasticity" is published.

## PLASTIC BEHAVIOUR FOR ONE- DIMENSIONAL LOADING



Direct strain in x direction is  $\epsilon_x$ .



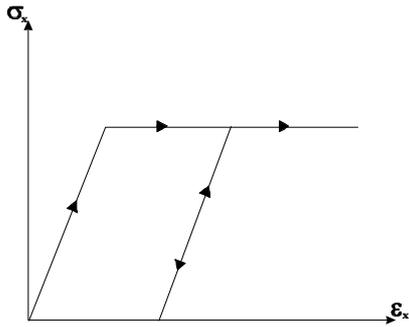
Typical stress -strain relation for an elastic, work hardening plastic material (e.g. metal alloy)

OA is elastic. A is a yield point.  $Y_1$  is the uniaxial yield stress.

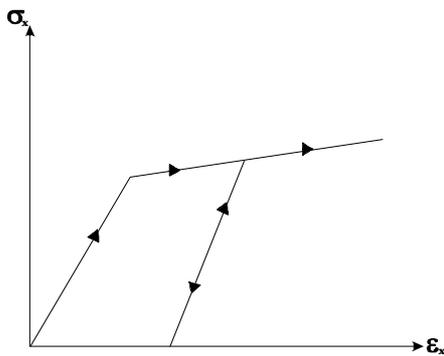
BC and DE are elastic unloading and reloading (parallel to OA).

On reloading to B the yield stress has increased to  $Y_2$ . The material is harder and has "strain (or work) hardened".

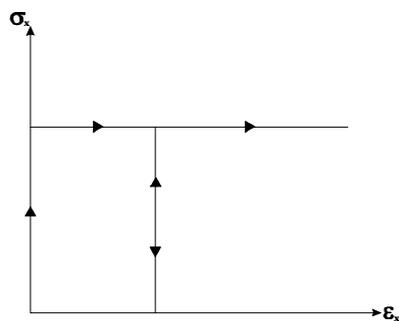
In describing plastic behaviour, the following simplifications (“idealisations”) are often made.



Elastic, perfectly plastic  
(steel behaviour is quite like this)

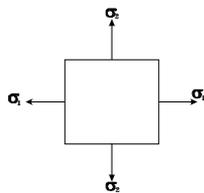


Elastic, linear strain hardening plastic



Rigid-plastic  
(often assumed in collapse calculations)

## YIELD FUNCTIONS - BASIC IDEAS



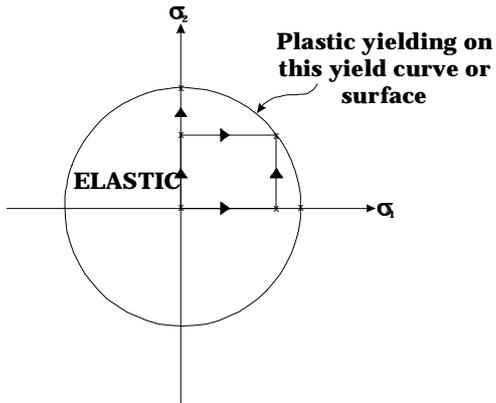
$\sigma_2$  is held constant and  $\sigma_1$  is increased until yielding starts (or vice versa)

The combinations of  $\sigma_1$  and  $\sigma_2$  that cause yielding are described by a yield function.

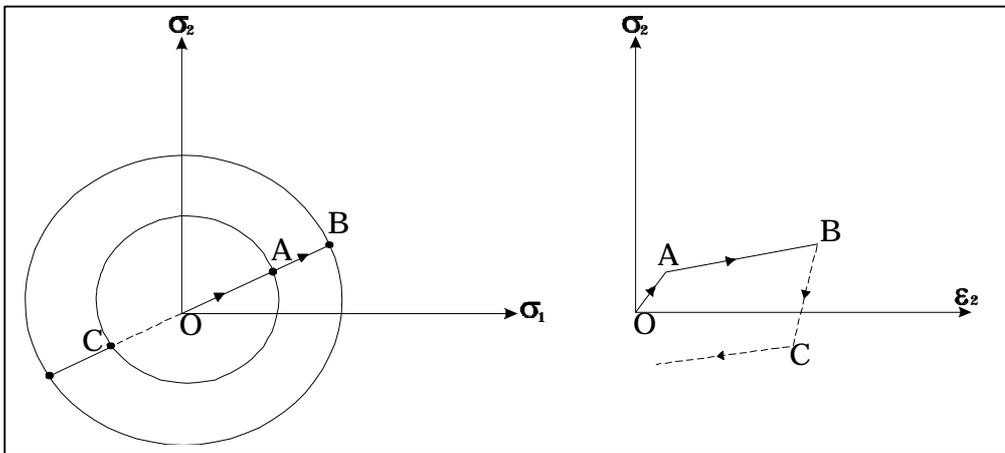
Yield functions can be represented by lines in 2D stress space and surfaces in 3D stress space.

For elastic-perfectly plastic behaviour:

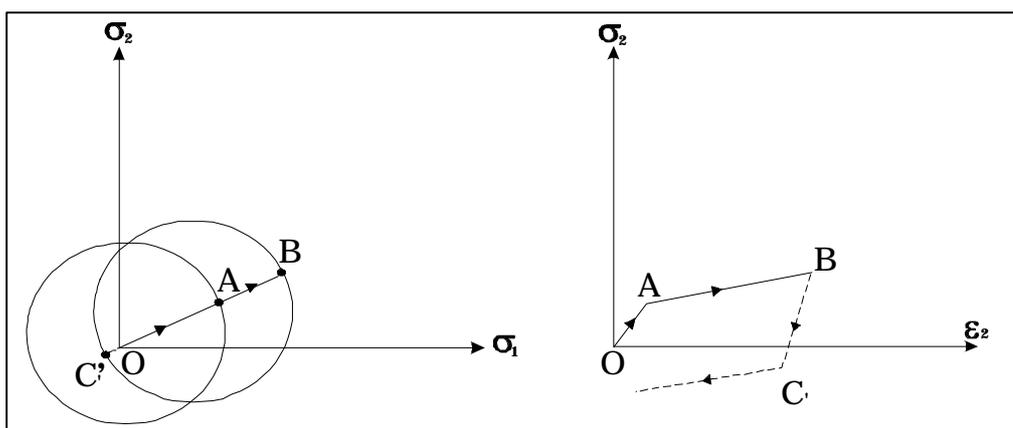
Stress states inside the yield surface are elastic.  
 Stress states outside the yield surface are impossible, by definition.



If strain hardening takes place, there are two possibilities:



The yield surface expands uniformly - this is called isotropic hardening



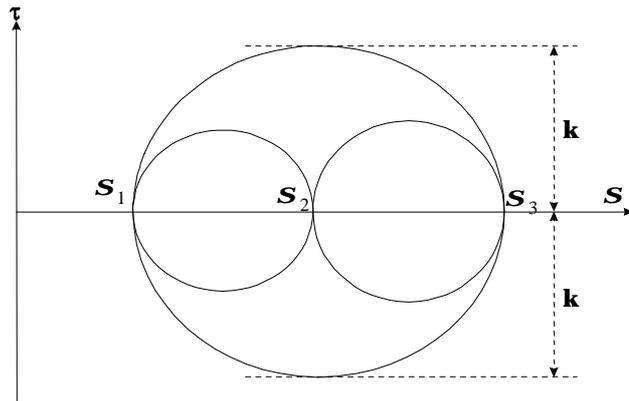
The yield surface is "dragged along" - this is called kinematic hardening.  
 The differences between these two assumptions become important if unloading takes place.

## YIELD FUNCTIONS - EXAMPLES

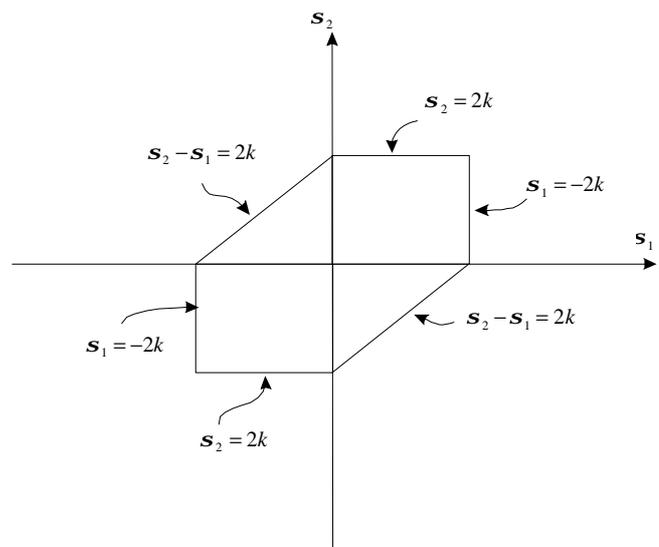
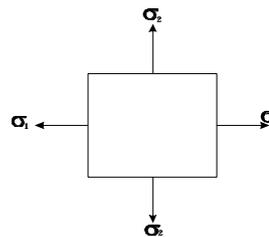
### a) TRESCA

Yielding takes place when the maximum value of  $|\sigma_1 - \sigma_2|$ ,  $|\sigma_2 - \sigma_3|$ ,  $|\sigma_3 - \sigma_1|$  equals a critical value ( $2k$ ).

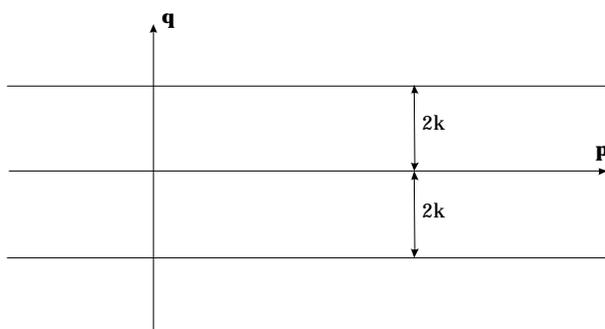
This can be interpreted (by considering Mohr's circles) as being equivalent to limiting the maximum shear stress on any plane in the material to being less than or equal to  $k$ .



Consider the yield condition in  $(\sigma_1, \sigma_2)$  stress space when  $\sigma_3 = 0$ .



For triaxial stress conditions  $\sigma_2 = \sigma_3$  and we can plot the yield function in  $(p, q)$  space.



b) VON MISES

Plastic yielding takes place when:

$$(\mathbf{s}_1 - \mathbf{s}_2)^2 + (\mathbf{s}_2 - \mathbf{s}_3)^2 + (\mathbf{s}_3 - \mathbf{s}_1)^2 = 2Y^2$$

where Y is the yield stress in uniaxial tension.

Note (from our earlier definition of q) that the above equation can be written as:

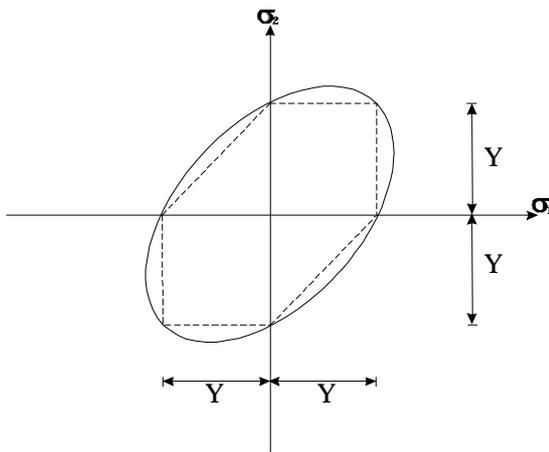
$$q = Y$$

Structural and mechanical engineers call the stress parameter q either:

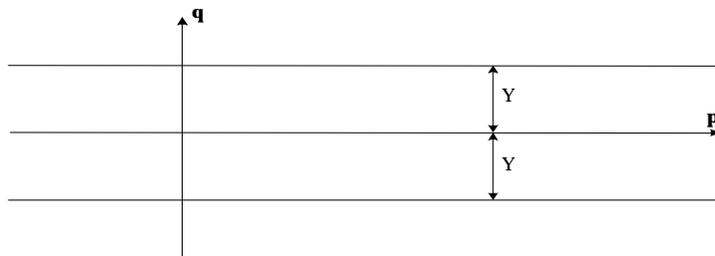
- i) the Von Mises stress,
- ii) the Von Mises equivalent stress, or
- iii) the equivalent stress

and use the symbol  $\sigma_E$  rather than q.

Von Mises yield function in  $(\sigma_1, \sigma_2)$  space when  $\sigma_3 = 0$  :

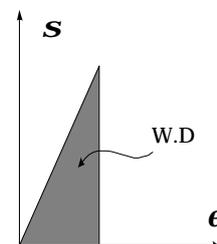


This is an ellipse with its major axis along  $\sigma_1 = \sigma_2$ . The Tresca yield criterion is shown thus ----- for comparison. Clearly  $Y = 2k$ . In triaxial stress conditions  $\sigma_2 = \sigma_3$  and the yield function in (p, q) space is as shown:



**STRAINS**

For every stress component or invariant there is a "corresponding" strain component so that the work done per unit volume in elastic deformation is =1/2 stress x strain.



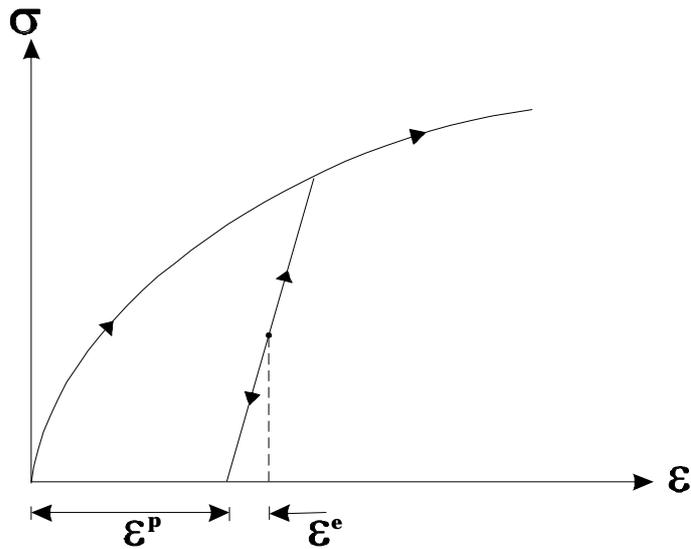
<u>Stress</u>	<u>Strain</u>
$\sigma_x$	$\epsilon_x$
$\sigma_y$	$\epsilon_y$
$\sigma_z$	$\epsilon_z$
$\tau_{xy}$	$\gamma_{xy}$
$\tau_{yz}$	$\gamma_{yz}$
$\tau_{zx}$	$\gamma_{zx}$
$\sigma_1$	$\epsilon_1$
$\sigma_2$	$\epsilon_2$
$\sigma_3$	$\epsilon_3$

<u>Stress invariant</u>	<u>Strain invariant</u>
<p><math>p</math> (defined previously)</p> $= \frac{\mathbf{s}_x + \mathbf{s}_y + \mathbf{s}_z}{3}$ $= \frac{\mathbf{s}_1 + \mathbf{s}_2 + \mathbf{s}_3}{3}$	<p><math>\epsilon_p</math> (volumetric strain)</p> $= \mathbf{e}_x + \mathbf{e}_y + \mathbf{e}_z$ $= \mathbf{e}_1 + \mathbf{e}_2 + \mathbf{e}_3$
<p><math>q</math> (defined previously)</p>	<p><math>\epsilon_q</math> (deviatoric strain)</p>

$$\mathbf{e}_q = \frac{2}{3} \sqrt{\frac{(\mathbf{e}_x - \mathbf{e}_y)^2 + (\mathbf{e}_y - \mathbf{e}_z)^2 + (\mathbf{e}_z - \mathbf{e}_x)^2 + \frac{3}{2}(\mathbf{g}_{xy}^2 + \mathbf{g}_{yz}^2 + \mathbf{g}_{zx}^2)}{2}}$$

$$\mathbf{e}_q = \frac{2}{3} \sqrt{\frac{(\mathbf{e}_1 - \mathbf{e}_2)^2 + (\mathbf{e}_2 - \mathbf{e}_3)^2 + (\mathbf{e}_3 - \mathbf{e}_1)^2}{2}}$$

## PLASTIC STRAINS



The plastic strain is the strain which remains on completely unloading the applied stress.

$$\epsilon^p + \epsilon^e = \epsilon^T$$

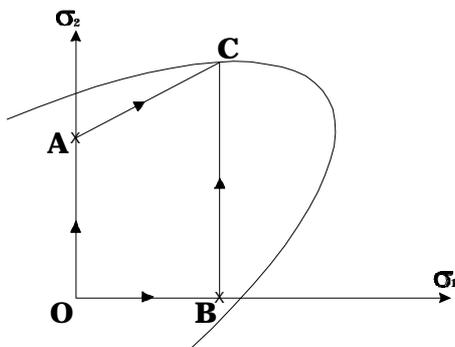
$\epsilon^p$  = plastic strain  
 $\epsilon^e$  = elastic strain  
 $\epsilon^T$  = total strain

## CALCULATION OF ELASTIC STRAINS

Suppose that the soil is in some stress state  $\underline{\sigma}$  ( $= [\sigma_1, \sigma_2, \sigma_3]^T$ ). An increment of stress is now applied to the soil ( $\Delta\underline{\sigma}$ ). The resulting strains, if the behaviour is elastic, can be calculated as  $\Delta\underline{\epsilon} = \mathbf{C}\Delta\underline{\sigma}$  where  $\mathbf{C}$  is a square matrix containing elastic constants.

## CALCULATION OF PLASTIC STRAINS

In contrast to elastic behaviour, it is found that plastic strains are strongly dependent on the current total stresses.



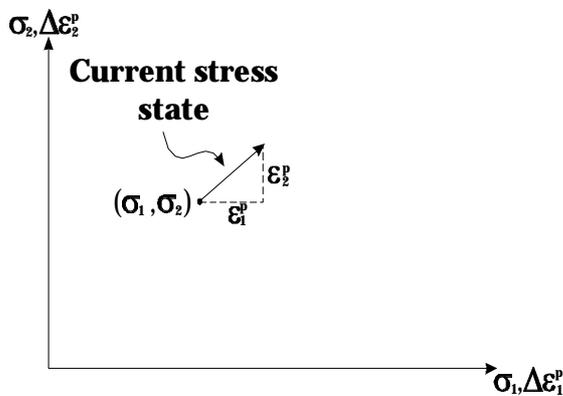
Thus the plastic strains generated a yield point C are not dependent on the direction of the stress path when yielding starts.

i.e. paths OAC and OBC lead to the same plastic strains.

# PLASTIC POTENTIALS

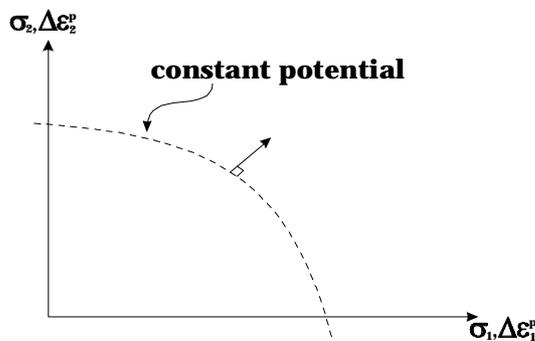
Mathematically, the type of behaviour described above can conveniently be expressed in terms of a potential function such that the derivatives of the potential function define the ratios of the plastic strains.

Plastic strain components are often plotted in stress space:



often called the strain increment vector

Mathematically we write:



$$d\mathbf{e}^p = m \frac{\nabla G}{\|\mathbf{s}\|} \quad (1)$$

where G is the plastic potential

For metals the plastic potential function is the same as the yield function i.e.  $G \equiv F$  and hence we can write:

$$d\mathbf{e}^p = m \frac{\nabla F}{\|\mathbf{s}\|} \quad (2)$$

(m is a scalar number which depends on the hardening law and the details of a particular analysis. It is called the plastic multiplier).

when  $G \equiv F$  we say there is normality (the strain increment vector is normal to the yield surface) - also called associated flow.

Equations like (1) and (2) above are called flow rules.

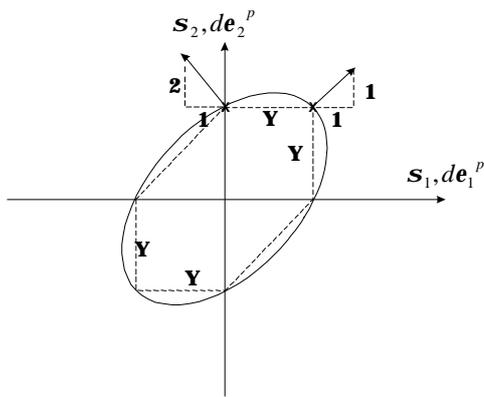
They govern the ratios of the plastic strain components.

## Flow Rule - Example with Von Mises

(a) in  $\sigma_1, \sigma_2$  space when  $\sigma_3 = 0$

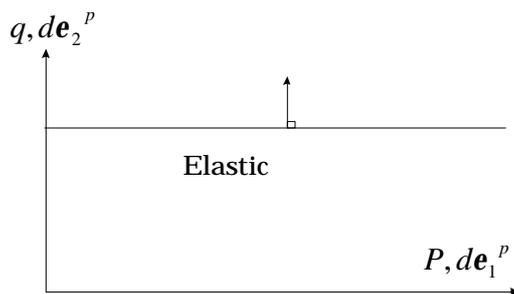
$$F(\mathbf{s}_1, \mathbf{s}_2) = \mathbf{s}_1^2 + \mathbf{s}_2^2 - \mathbf{s}_1 \mathbf{s}_2 - Y^2 = 0$$

$$\begin{bmatrix} d\mathbf{e}_1^p \\ d\mathbf{e}_2^p \end{bmatrix} = \begin{bmatrix} m \frac{\partial F}{\partial \mathbf{s}_1} \\ m \frac{\partial F}{\partial \mathbf{s}_2} \end{bmatrix} = \begin{bmatrix} m(2\mathbf{s}_1 - \mathbf{s}_2) \\ m(2\mathbf{s}_2 - \mathbf{s}_1) \end{bmatrix}$$



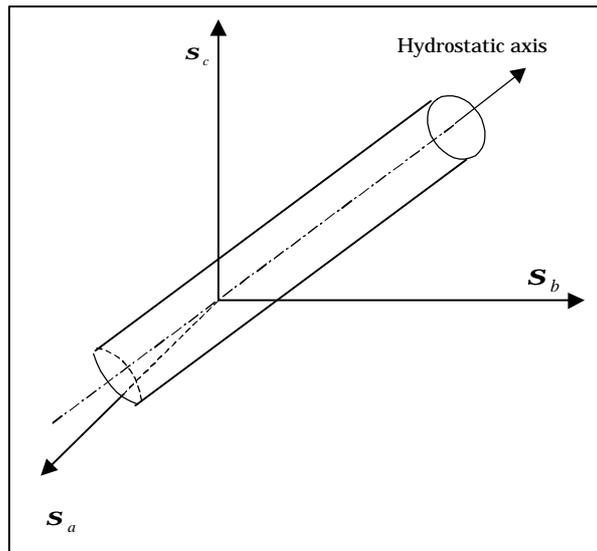
By inspection the strain increment vector is perpendicular to the ellipse previously described.

(b) in  $p, q$  space

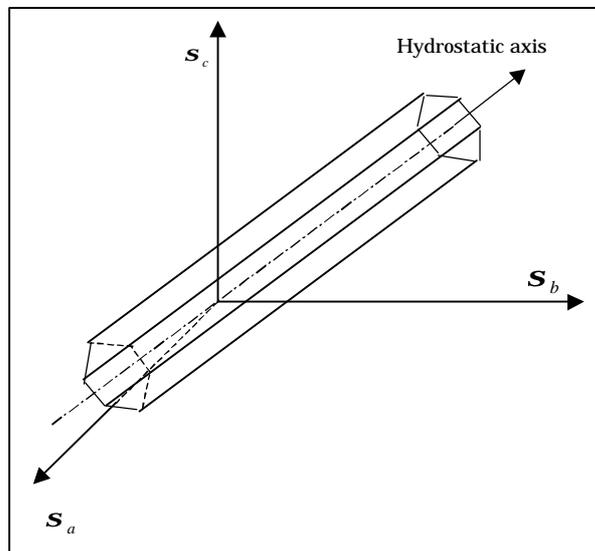


Yielding takes place with zero volumetric strains  
 $(d\mathbf{e}_p^p = 0)$

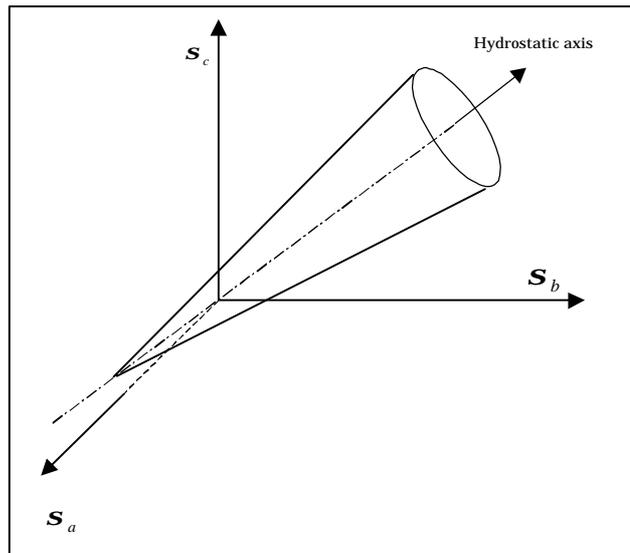
(1) The Von Mises Yield Surface



(2) The Tresca Yield Surface



(3) The Drucker-Prager Yield Surface



(4) The Mohr-Coulomb Yield Surface

